

# Consistent coupling of positions and rotations for embedding 1D Cosserat beams into 3D solid volumes



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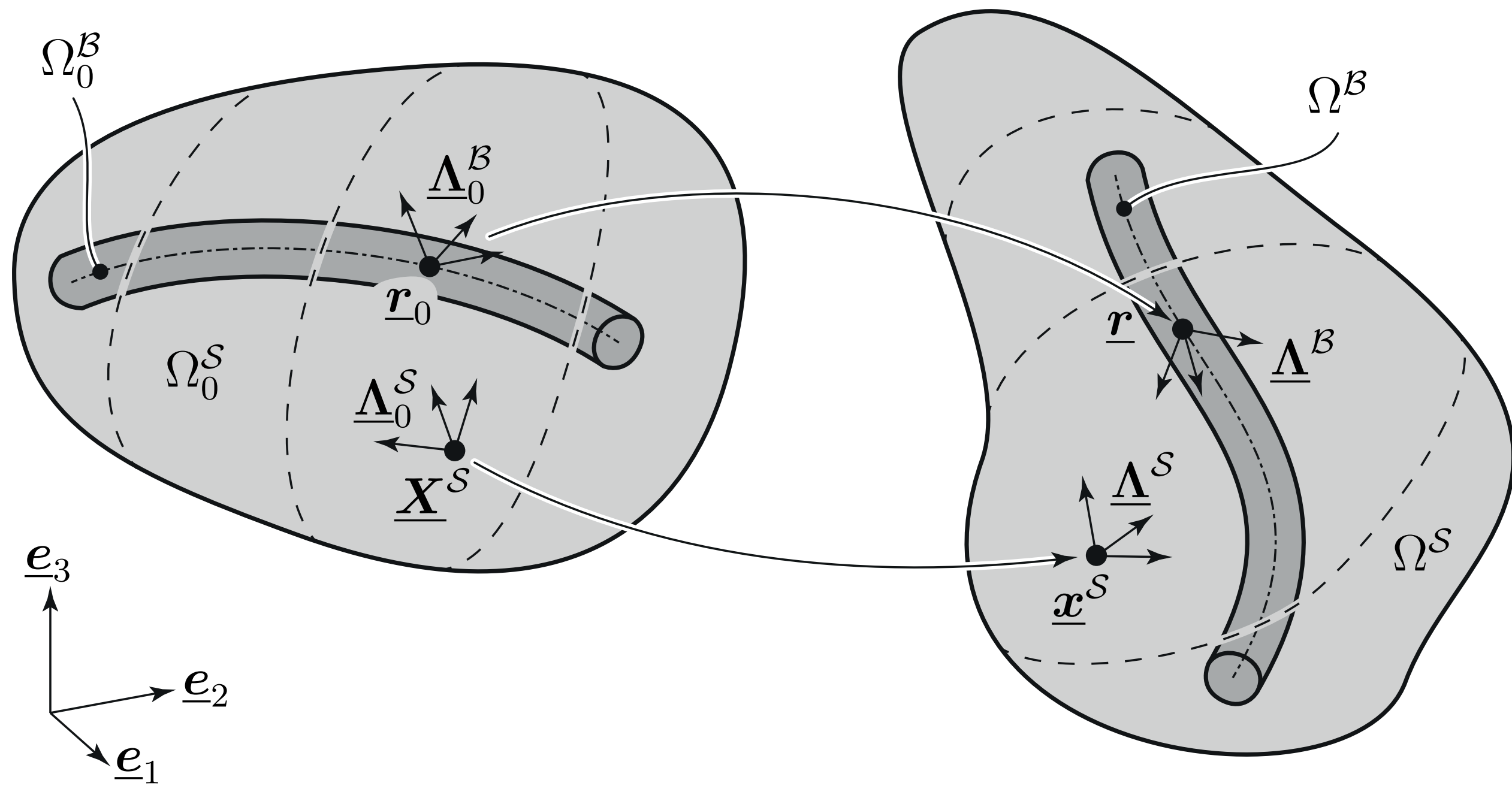
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## Motivation

Embedding 1D Cosserat continua (fibers / beams) [1] into 3D solid volumes requires a consistent method to model the mixed-dimensional coupling interactions between the two domains. Mechanically, the interaction between the fibers and the solid volume is defined on the 2D surface of the fibers, i.e., a 2D-3D coupling.



We approximate the 2D-3D coupling by a 1D-3D coupling along the centerline curve of the fiber  $\Gamma_c$  via,

$$\begin{aligned} \mathbf{r} - \mathbf{x}^S &= \mathbf{0} \quad \text{on } \Gamma_c \\ \underline{\psi}_{SB} &= \mathbf{0} \quad \text{on } \Gamma_c. \end{aligned}$$

The first set of constraint equations describes the coupling of the positions along the fiber centerline [2]. Only coupling the positions can lead to an underestimated stiffness of the compound structure, therefore, the second set of constraint equations enforces a coupling of the fiber cross-section rotations and the solid volume. This rotational coupling will be investigated here.

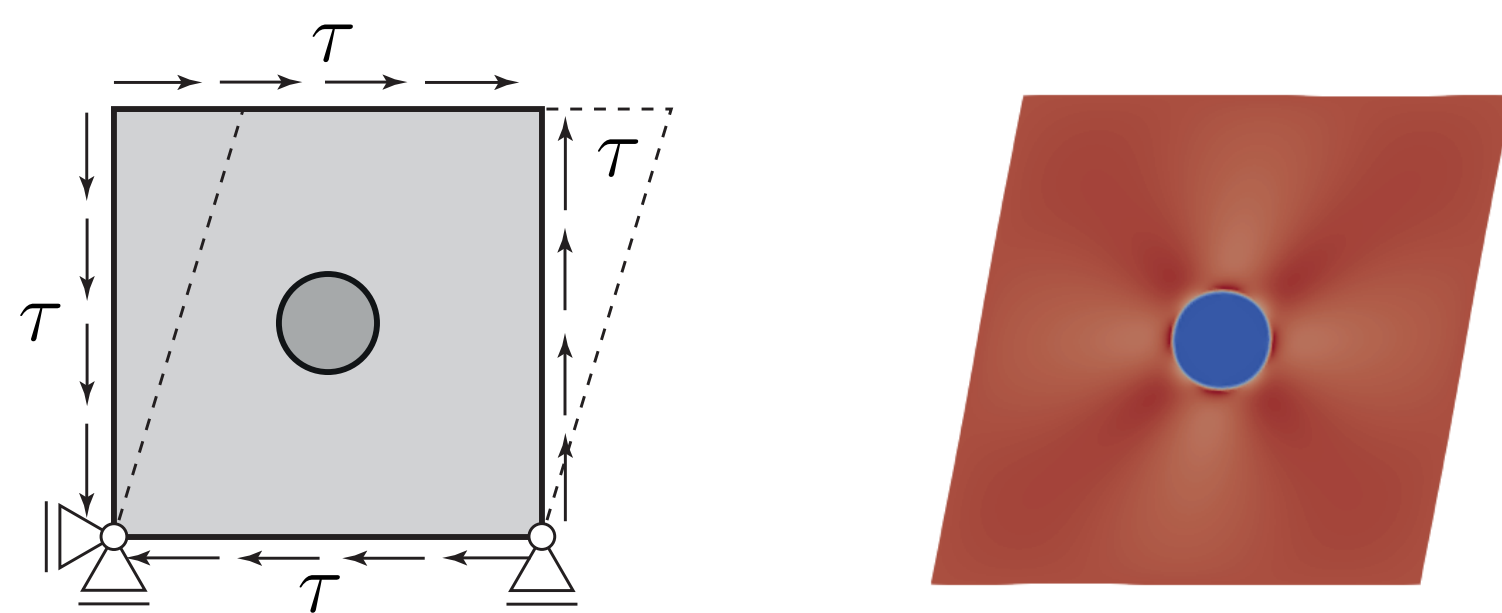
## Solid Triad Field

The rotational coupling constraints enforce a vanishing relative (pseudo-)rotation vector between a beam cross-section triad  $\underline{\Delta}^B$  and a corresponding solid triad  $\underline{\Delta}^S$ , cf. [3],

$$\underline{\psi}_{SB} = \text{rv} \left( \underline{\Delta}^S \underline{\Delta}^{B^T} \right) = \mathbf{0}.$$

One main contribution of this work is the definition of a suitable solid triad field in the solid (Boltzmann) continuum.

A **pure shear** problem is considered as a benchmark example (with a fine reference solution)

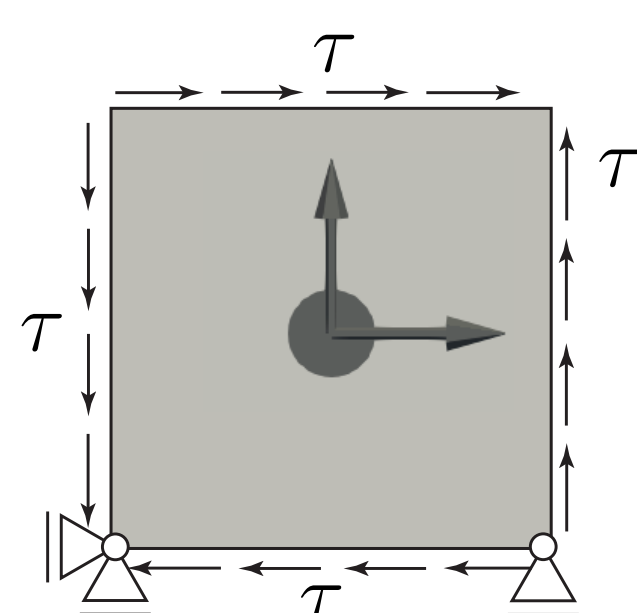


A first order Taylor series expansion of the 2D-3D coupling leads to the following solid triad field:

$$\underline{\Delta}^S = \underline{\mathbf{F}}$$

with the the solid deformation gradient  $\underline{\mathbf{F}}$

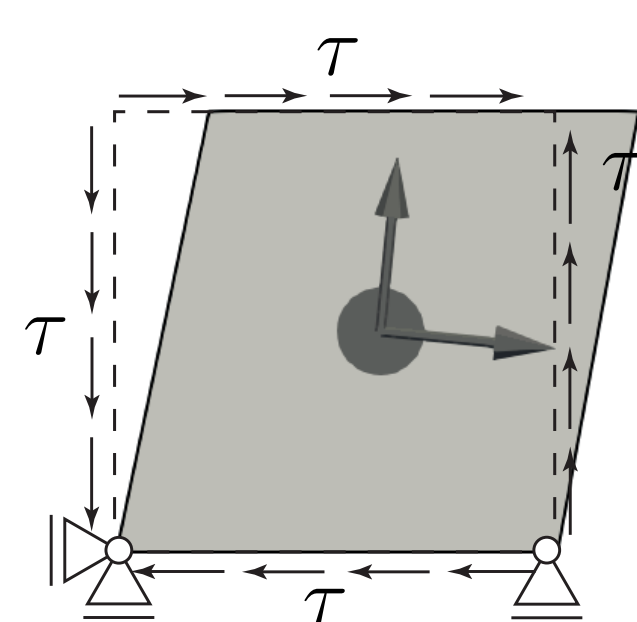
**Catastrophic shear locking!**



As an alternative, the solid triad can be obtained as the rotational part of the polar decomposition of the solid deformation gradient  $\underline{\mathbf{F}} = \underline{\mathbf{R}}\underline{\mathbf{U}}$ :

$$\underline{\Delta}^S = \underline{\mathbf{R}}$$

**Very good agreement with the reference solution!**



## Discretization

- The rotational coupling constraints are enforced via the Lagrange multiplier method

$$\delta \Pi_{\lambda}^{\mathcal{R}} = \int_{\Gamma_c} \delta \underline{\lambda}^{\mathcal{R}^T} \underline{\psi}_{SB} \, ds + \int_{\Gamma_c} \underline{\lambda}^{\mathcal{R}^T} \delta_o \underline{\psi}_{SB} \, ds$$

- A mortar-type approach is used for the spatial discretization of the Lagrange multiplier field

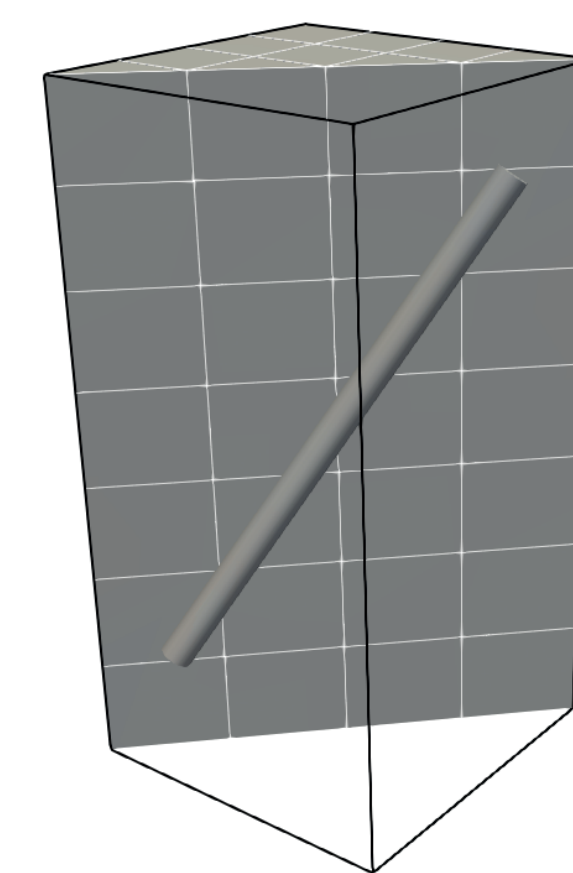
$$\underline{\lambda}_h^{\mathcal{R}(f)} = \sum_{j=1}^{n^{\mathcal{R}(f)}} \Phi_j^{\mathcal{R}(f)}(\xi^B) \hat{\underline{\lambda}}_j^{\mathcal{R}(f)}$$

- Global system of equations resulting from mortar-type discretization of **rotational** coupling and **positional** coupling (according to [2])

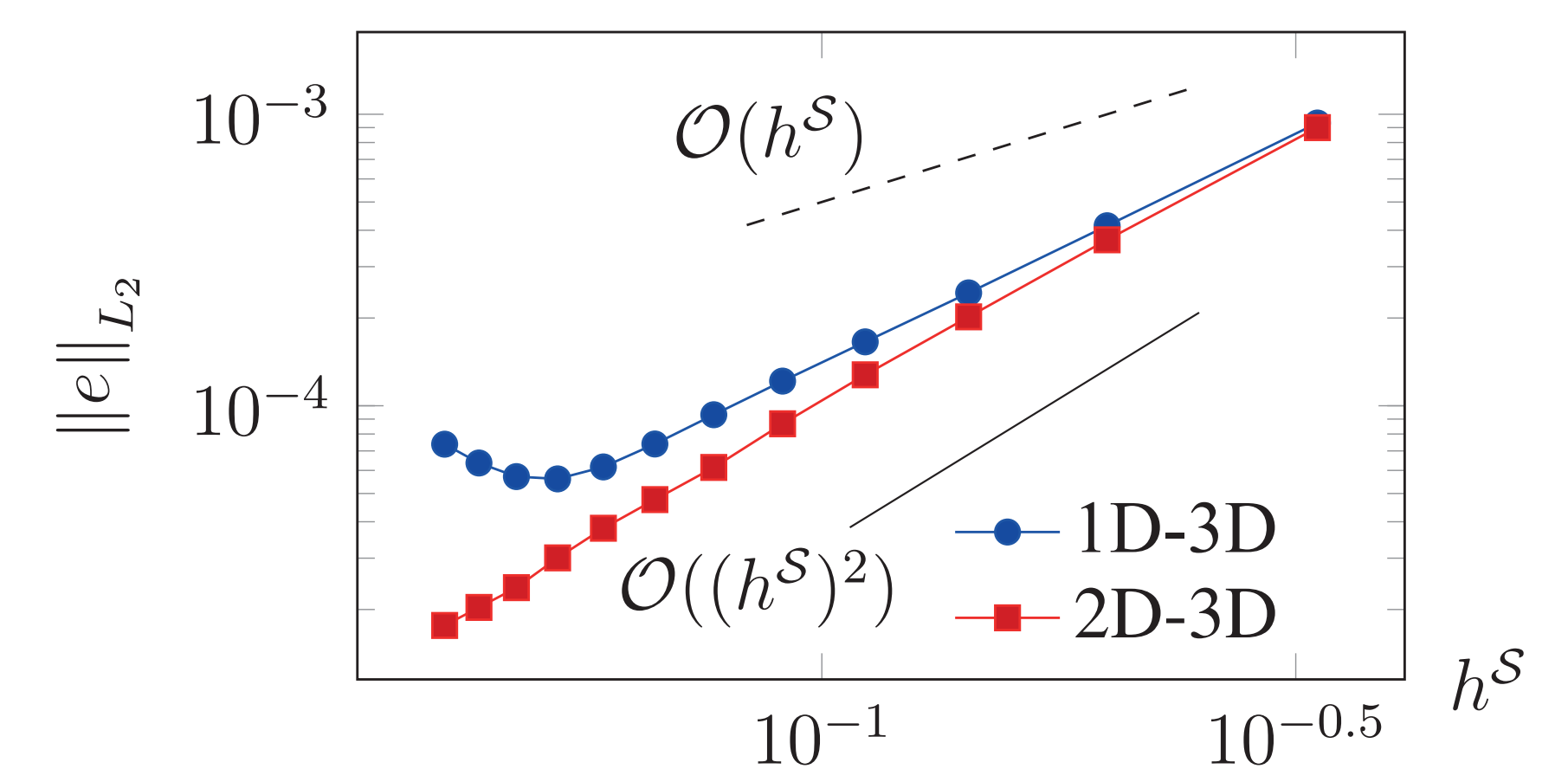
$$\begin{bmatrix} \mathbf{K}_{ss}^S + \mathbf{Q}_{ss}^{\mathcal{R}} & \mathbf{0} & \mathbf{Q}_{s\theta}^{\mathcal{R}} & -\mathbf{M}^{\nu^T} & \mathbf{Q}_{s\lambda}^{\mathcal{R}} \\ \mathbf{0} & \mathbf{K}_{rr}^B & \mathbf{K}_{r\theta}^B & \mathbf{D}^{\nu^T} & \mathbf{0} \\ \mathbf{Q}_{\theta s}^{\mathcal{R}} & \mathbf{K}_{\theta r}^B & \mathbf{K}_{\theta\theta}^B + \mathbf{Q}_{\theta\theta}^{\mathcal{R}} & \mathbf{0} & \mathbf{Q}_{\theta\lambda}^{\mathcal{R}} \\ -\mathbf{M}^{\nu} & \mathbf{D}^{\nu} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{Q}_{\lambda s}^{\mathcal{R}} & \mathbf{0} & \mathbf{Q}_{\lambda\theta}^{\mathcal{R}} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{bmatrix} \Delta \mathbf{d}^S \\ \Delta \mathbf{d}^B \\ \Delta \boldsymbol{\theta}^B \\ \boldsymbol{\lambda}^{\nu} \\ \boldsymbol{\lambda}^{\mathcal{R}} \end{bmatrix} = \begin{bmatrix} -\mathbf{r}^S \\ -\mathbf{r}_r^B \\ -\mathbf{r}_\theta^B \\ -\mathbf{g}^{\nu} \\ -\mathbf{g}^{\mathcal{R}} \end{bmatrix}$$

## Numerical Examples

consistency / verification

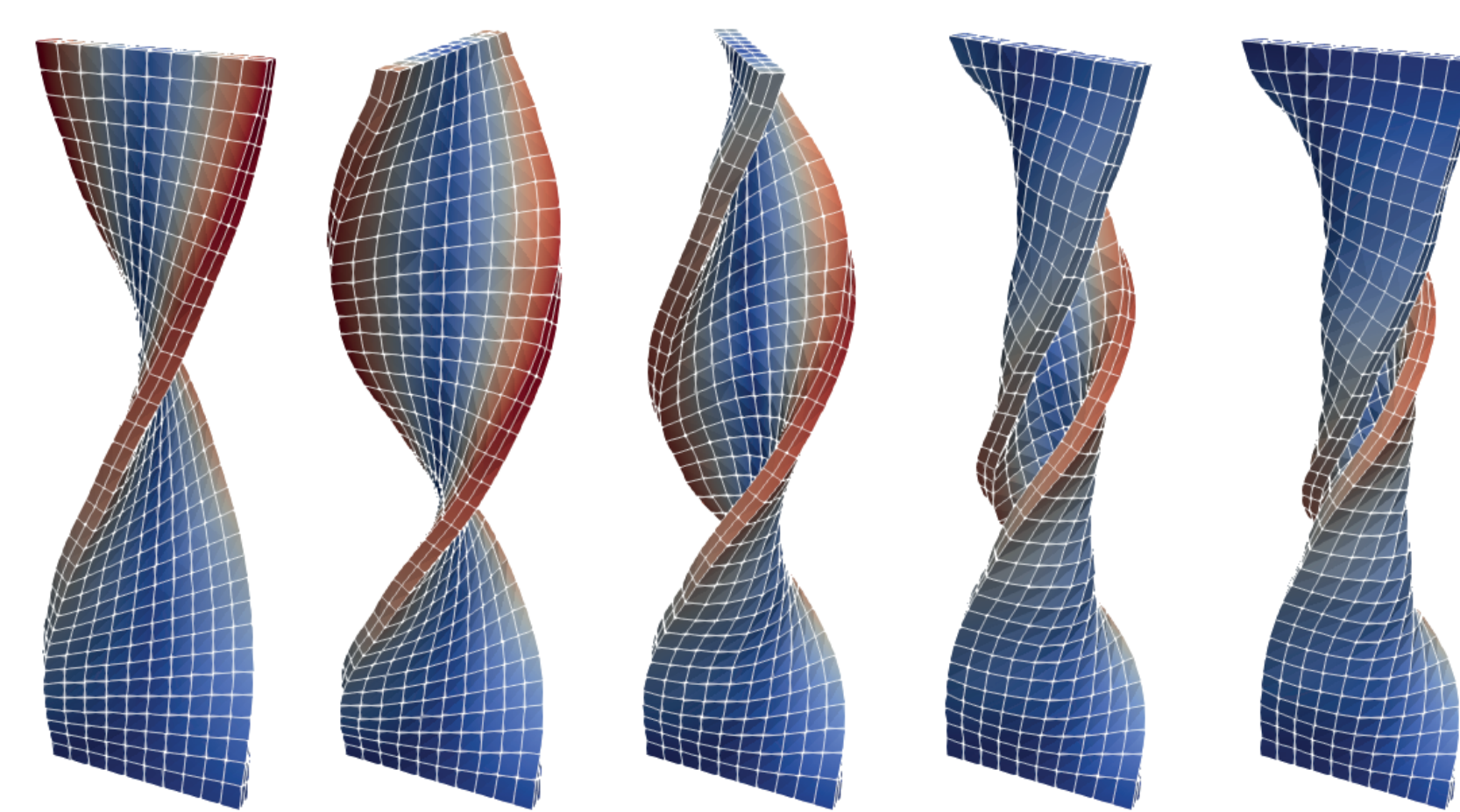


Patch tests are fulfilled



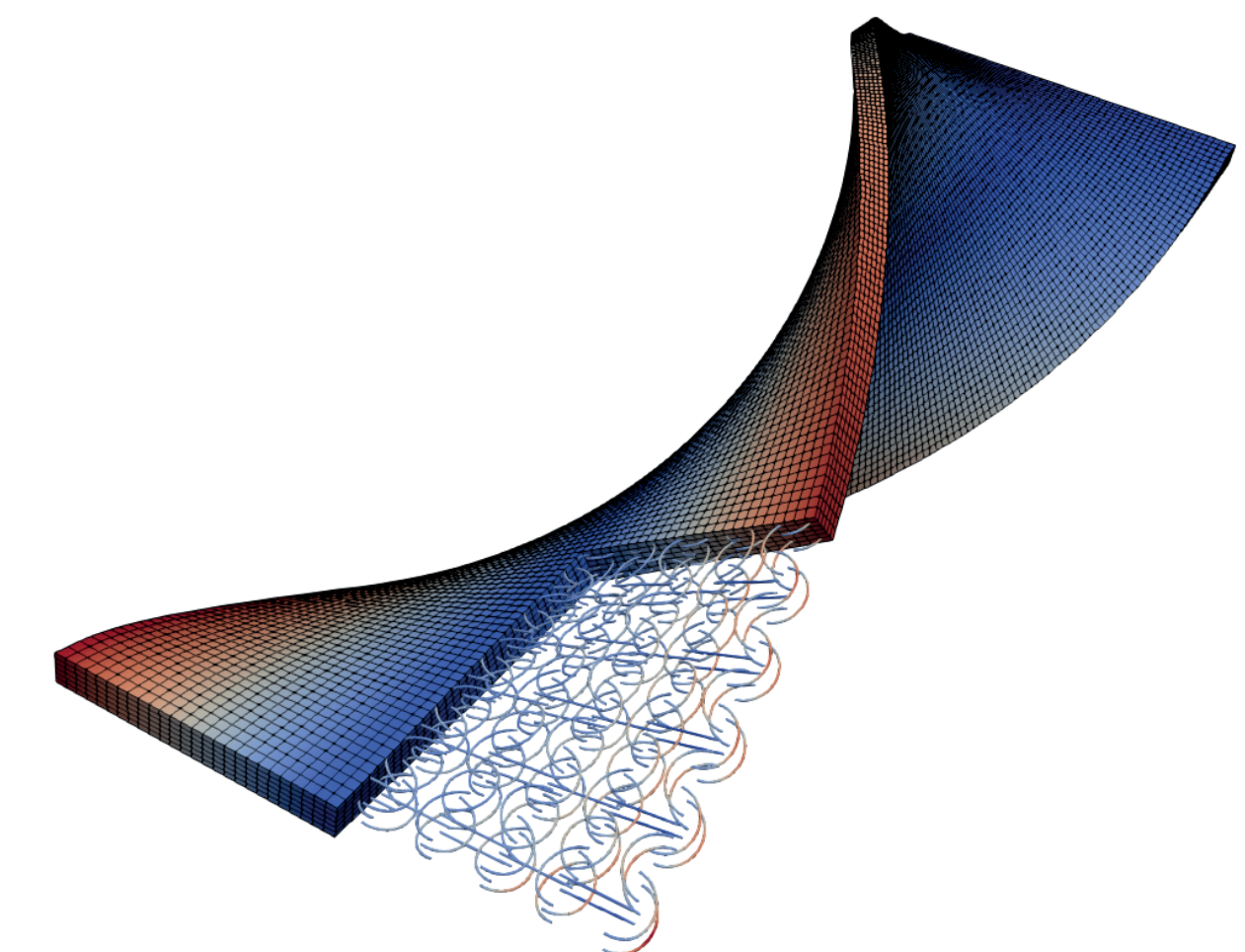
Spatial convergence for reasonable beam to solid element size ratios

applications / validation



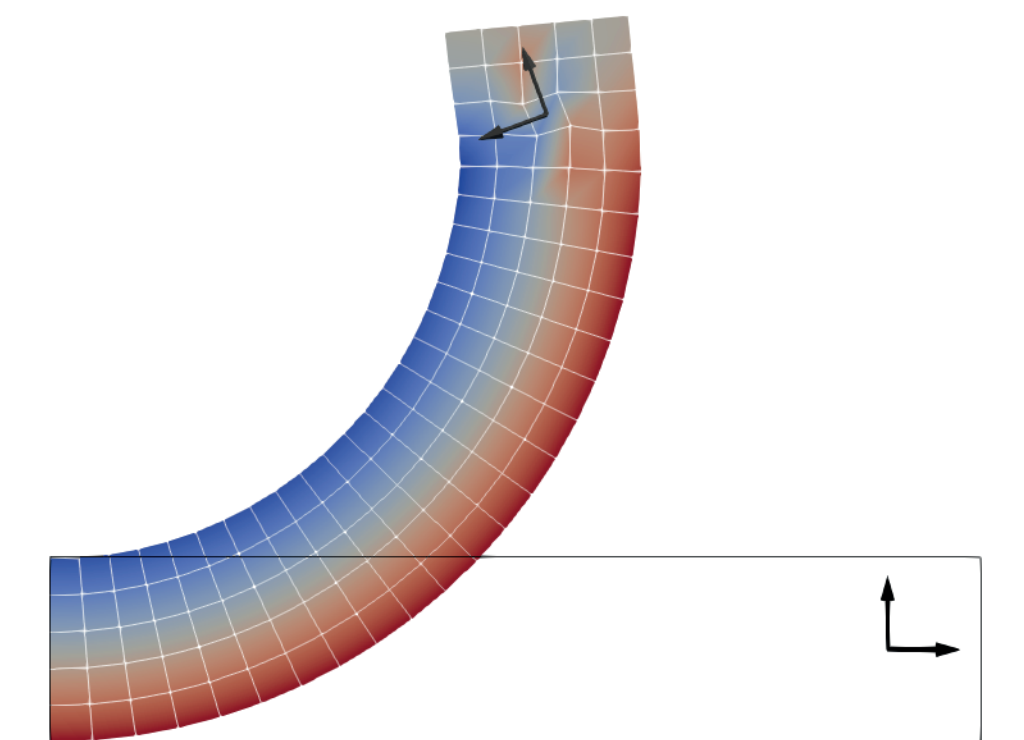
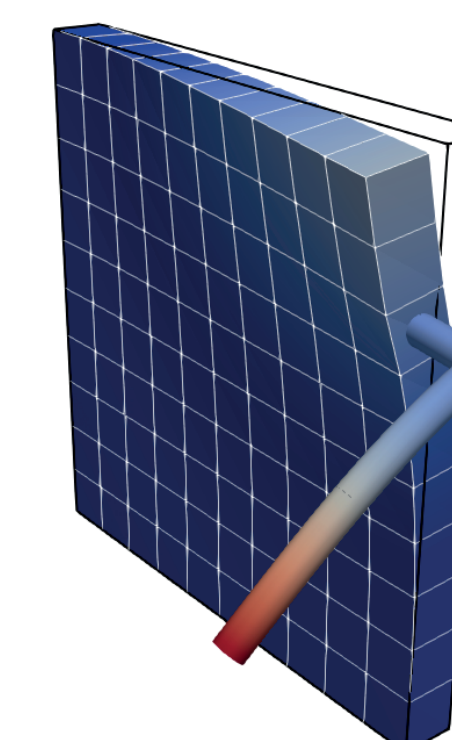
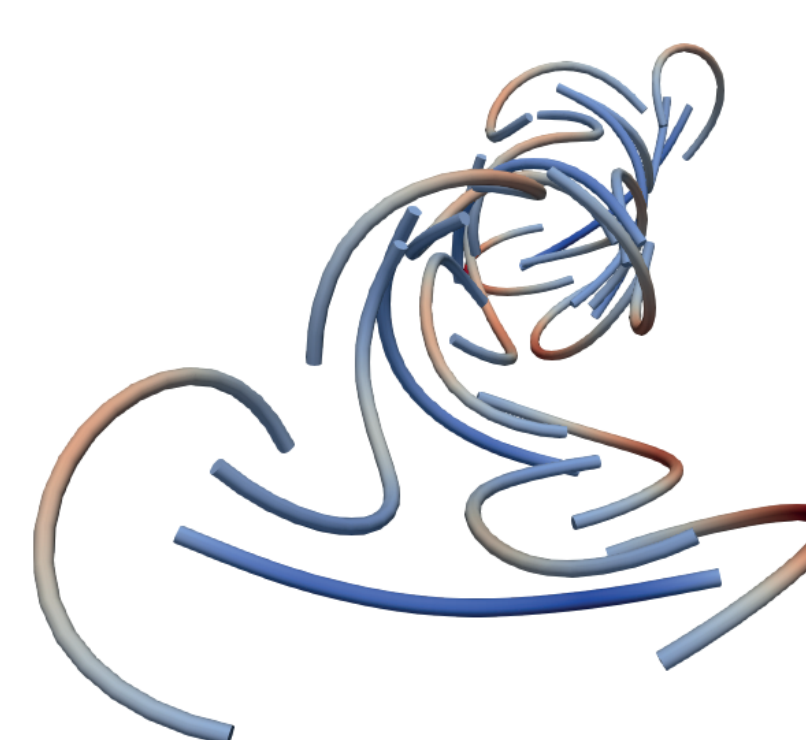
**Twisted composite plate**

Only positional *and* rotational coupling results in a correct stiffness of the compound structure



**Large scale composite plate**

The presented method is suitable for large scale applications



## References

- [1] Meier, C., Popp, A., Wall, W.A.: Geometrically exact finite element formulations for slender beams: Kirchhoff–Love theory versus Simo–Reissner theory. *Archives of Computational Methods in Engineering* **26**(1), 163–243 (2019)
- [2] Steinbrecher, I., Mayr, M., Grill, M.J., Kremheller, J., Meier, C., Popp, A.: A mortar-type finite element approach for embedding 1D beams into 3D solid volumes. *Computational Mechanics* **66**(6), 1377–1398 (2020)
- [3] Meier, C., Grill, M.J., Wall, W.A.: Generalized section-section interaction potentials in the geometrically exact beam theory (2021). Preprint, <https://arxiv.org/abs/2105.10032>